



## MAHARASHTRA STATE BOARD OF TECHNICAL EDUCATION (Autonomous)

(ISO/IEC -270001 – 2005 certified)

## **WINTER -2019 EXAMINATION**

**SUBJECT CODE:** 

22402

## MODEL ANSWER

## **Important Instructions to examiners:**

- 1) The answer should be examined by keywords and not as word-to-word as given in the model answer scheme.
- 2) The model answer and the answer written by candidate may vary but the examiner may try to assess the understanding level of the candidate.
- 3) The language error such as grammatical, spelling errors should not be given more importance.
- 4) While assessing figures, examiner may give credit for principal components indicated in the figure. The figure drawn by candidate and model answer may vary. The examiner may give credit for any equivalent figure drawn.
- 5) Credits may be given step wise for numerical problems. In the some cases, the assumed constants values may vary and there may be some difference in the candidate's answer and model answer.
- 6) In case of some questions credit may be given by judgment on part of examiner of relevant answer based on candidates understanding.

Que. NO	Answer with question					
Q. 1	Attempt any FIVE of the following					
a)	Define core of section.					
Ans.	Core of a section: Core of the section is that portion around the centroid in within which the line of action of load must act, so as to produce only compressive stress is called as core of the section. It is also defined as the region or area within which if load is applied, produces only compressive resultant stress. If Compressive load is applied, the there is no tension anywhere in the section.  emax = d/8					
	e = Core of section  Core of section  For Circular section  For rectangular section	01 M				



b)	State the condition for no tension in the column section	
Ans.	Condition for no tension in the column section	
	$\sigma_0$ = Direct stress and $\sigma_b$ = Bending stress	
	,if $\sigma_0 > \sigma_b$ the resultant stress is compressive, If $\sigma_0 = \sigma_b$ the minimum stress is zero and the maximum stress is 260, the stress distribution is compressive. but $\sigma_0 < \sigma_b$ the stress is partly compressive and partly tensile. A small tensile stress at the base of a structure may develop tension cracks. Hence for no- tension condition, direct stress should be greater than or equal to bending	01 M
	stress. $\sigma_0 > = \sigma_b$ P / A = M/Z P / A = Pxe/Z , e = < Z/A Hence for no -tension condition, eccentricity should be less than Z/A	01 M
c)	State expression for deflection of simply supported beam carrying point load	
Ans.	at midspan.  A simply supported beam of span L carrying a central point load F at	
	midspan  A  B	01 M
	To find the maximum deflection at mid-span, we set x = L/2 in the equation and obtain ,maximum deflection = Ye	01 M
<b>d</b> )	Yc = Y max = F L <sup>3</sup> / 48 El State the values of maximum slope and maximum deflection for a cantilever	U1 IVI
u)	beam of span 'L' carrying a point load 'W' at the free end . EI = constant	
Ans.	$A = \frac{1}{2} $	01 M



<b>e</b> )	Compare a simply supported beam and a continuous beam w.r.t deflected	
	shape of a beam.	
Ans.	The firm of a curve to which the longitudinal axis of the beam bends after loading is called elastic curve or deflected shape of the beam. In the figure shows the deflected shape for various types of continuous beam. The deflected shape is shown by a dotted curve. Deflected shape simply supported beam and continuous beam  (i) Continuous beam with simply supported ends  W  W  (i) Continuous beam with simply supported ends	01 M
	(ii) Continuous beam with one end fixed and other simply supports	
	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	O1 M (Any one sketc h)
	(iv) Continuous beam with end span overhanging	
	(a) seminated beam with one of	
<b>f</b> )	Write the values of stiffness factor for beams.	
	i) Simply supported at both ends	
	ii)/fixed at one end simply supported at other end	
Ans.	i)Stiffness factor for a beam Simply supported at both the ends = 3EI/L	01 M
	ii) Stiffness factor for a beam fixed at one end and simply supported at other end = 4EI/L	01 M
<b>g</b> )	Make the following truss perfect by adding or removing the members, if	
	required as shown in fig. No.1	
	D .	
	B c A C B	
	(ii) (ii) B	
	- agr - 130 - 2	



Ans.	For i) n=5 ,j=4	
7113	$2j-3 = 2 \times 4 - 3 = 5$ .since $n = 2j-3$ hence the frame is Perfect frame  iii) $n = 5$ , $j=4$ , $2j-3 = 2 \times 4 - 3 = $ since $n = 2j-3$ hence the frame is Perfect frame	01 M
	A (ii)	01 M
Q. 2	Attempt any THREE of the following:	12 M
a)	Explain the effect of eccentric load with sketch w.r.t stresses developed	
Ans.	Effect of eccentric load: A load whose line of action does not coincide with the axis of a member is called an eccentric load. The distance between the eccentric axis of the body and the point of loading is called an eccentric limit 'e'. Due to effect of eccentricity axial load causes only direct stress whereas an eccentric load causes direct as well as bending stresses. Direct load is that force which act a centroidal longitudinal axis of the member. Eccentric load is that force which act away from centroidal longitudinal axis of the member. Thus the resultant stresses due to direct as well as bending stresses in the member	02 M 01 M
	(ii) Plan	
	Direct stress = $\sigma 0$ , Bending stress = $\sigma b$	
	$\sigma 0 = P / A$ , $\sigma b = (M \times y) / I$ therefor $\sigma b = M/Z$ But, Resultant stresses =	
	$\sigma_{\text{direct}} + \sigma_{\text{bending}} \sigma_{\text{max}} = 60 + 6b$ ,	01 M
	$\sigma_{\min} = 60 - 6b$	
<b>b</b> )	Explain with expression four conditions of stability of dam.	
Ans.	1. Condition to prevent Overturning of a dam Stability against Due to  Overturning (P.h/3) < W(b-X)	01 M



	2. Condition to prevent sliding of a dam ,Stability against Due to	01 M
	Sliding $P \le F P \le \mu$ W factor of safety against sliding	OT M
	3. Compression or Crushing of masonry	01 M
	4. Condition to avoid tension in the masonry Stability against No Tension if	
	e < (b/6) Where $e =$ eccentricity	01 M
,	P = Compressive Load h = Ht. of dam W = Wt of dam b = Base width of dam	
<b>c</b> )	Calculate maximum and minimum stresses at base of a rectangular column as shown in Fig No.2 . It carries a load 200 KN at 'P' on the outer edge of a column. Draw stress distribution diagram. $Y$	
	X 100mm 200mm	
Ans.	Solution :-	
	Area =200 x 100 = 20000 mm <sup>2</sup> P = 200kN e = 50 mm M = P x e = 200 x 50 =10000 kN mm I =bd <sup>3</sup> /12 = 200x100 <sup>3</sup> /12 = 16.66x10 <sup>6</sup> = mm <sup>4</sup> y = 100/2 = 50 mm. Where, Stresses i) $6_0$ = P / A = 200x $10^3$ / $20000$ = 10 N/ mm <sup>2</sup>	01 M
	ii) $6b = (M \times y) / I$	0.1.7.7
	$(10000 \times 10^3) \times 50 / 16.66 \times 10^6 = 30.012 \text{ N/ mm}^2$ But, $6\text{max} = 60 + 6\text{b}$ , $6\text{min} = 60 - 6\text{b}$	01 M
	$6\text{max} = 6_0 + 6b = 10 + 30.012 = 40.012 \text{ N/mm}^2$	
	$6\min = 6_0 - 6b = 10 - 30.012 = -20.012 \text{ N/mm}^2 \text{ (Tension)}$	01 M



	stress distribution diagram as below	
	o <sub>min</sub> + o <sub>max</sub>	01 M
	Stress distribution diagram at base	
d)	Calculate the values of direct stress and bending stress at the base of chimney. Write interpretation of obtained values of stresses.  Use following data	
	i) External diameter = 3m	
	ii) Internal diameter = 2m	
	iii) Height of chimney = 44m	
	iv) Weight of masonry = 20 kN/m2	
	v) Co-efficient of wind resistance = 0.60 vi)Wind pressure = 1 kN/m2	
Ans.		
	Solution:	
	Given = $d1=3m$ , $d2=2m$ , height of chimney h =44	
	i) Area of the section = $A = (\pi/4) \times (3^2 - 2^2) = 3.926 \text{ m}^2$	
	$I xx = I = \pi /64 (3^4 - 2^4) = 51.05 \text{ mm}^4$	
	Wind pressure $\neq P = 1 \text{ kN/m}^2 = 1000 \text{ N/m}^2$	
	ii) Direct stress on the base $\sigma_0 = W / A$	
	$= A x h x \rho = (3.926 x 44 x 20) / A$	01 M
	$=880 \text{ kN/m}^2$	UI IVI
	iii) section modulus $Z = \pi / 32 \times (3^4 - 2^4) / 3 = 2.127 \text{ m}^3$	
	iv) Total wind load P = C x P x projected area	
	iv) Total wind load P = C x P x projected area = 0.6 x P x D x h = 0.6 x 1 x 3 x 44 = 79.2	
	iv) Total wind load P = C x P x projected area = 0.6 x P x D x h = 0.6 x 1 x 3 x 44 = 79.2 v) Moment on the base M= P x h/2 = 79.2 x 44 /2 = 1742.40 kNm	
	iv) Total wind load P = C x P x projected area = 0.6 x P x D x h = 0.6 x 1 x 3 x 44 = 79.2	01 M
	iv) Total wind load $P = C \times P \times P$ projected area $= 0.6 \times P \times D \times D$	01 M 01 M



	1/00/10 3/1 2	
	<b>1699.18</b> N/mm <sup>2</sup>	
		01 M
	$60.82 \text{ N/mm}^2$	
	Stress distribution diagram at base	
3.	Attempt any THREE of the following	12 M
	Calculate the deflection under point load of a simply supported beam as shown	
	in figure No. 3 Take EI = constant. Use Macaulay's method.	
	9 KN II man till bla en de de	
<b>a</b> )	AND THE PROPERTY OF THE PARTY OF	
	A B	
	1m 2m	
	Figure 3	
Ans:	(i) (ii)	
	2 gKN	
	A IVC I	
	1m 2 m	
	$\chi$	
	RA=6KN RB=3KN	
	Taking moment at B $\sum M_B = 0$	
	$R_A \times 3 - 9 \times 2 = 0$	
	$R_A = 6 \text{ kN}$ . And $R_B = 3 \text{ kN}$	
	Macaulay's method	
	$EI \frac{d^2y}{dx^2} = M$ Differential Equation	
	$EI\frac{d^2y}{dx^2} = 6x - 9(x-1)$	01 M
	$dx^*$	
	N=1 Differentiating with respect to y	
	Differentiating with respect to $x$	
	$EI\frac{dy}{dx} = \frac{6x^2}{2} + C_1 - \frac{9(x-1)^2}{2}$ Slope Equation	
	ax z	
	2.13	
	EIy = $\frac{3x^3}{3} + C_1x + C_2 - \frac{9(x-1)^3}{6}$ Deflection Equation	
	3 6	
	Calculate Constants of Integration $C_1$ and $C_2$	
	Consider boundary condition	



	1) At $x=0$ , $y=0$ putting in deflection equation	
	EI (0) = $0 + C_1 \times 0 + C_2$	
	$\mathbf{C_2} = 0$	
	2) At $x = 3m$ , $y = 0$ putting in deflection equation	01 M
	EI (0) = $3^3 + 3 C_1 + 0 - \frac{9}{6}(3-1)^3$	UI IVI
	$C_1 = -5$	
	Putting values of C <sub>1</sub> and C <sub>2</sub> in Slope and Deflection Equation.	
	$EI\frac{dy}{dx} = \frac{6x^2}{2} - 5 - \frac{9(x-1)^2}{2}$ Final Slope Equation	
		01 M
	$EIy = \frac{3x^2}{3} - 5x - \frac{9(x-1)^2}{6}$ Final Deflection Equation	
	Calculate Deflection under point load	
	At $x = 1m$ , $y = y_c$ putting in deflection equation.	
	EI $y_c = \frac{3(1)^3}{3} - 5(1) - 9(0)$	01 M
		V = -:-
	$y_c = \frac{-4}{EI}$	
	Calculate fixed end moments and draw BMD for a fixed beam as shown in Fig.	
<b>b</b> )	20kN 32kN  A 2m D 2m B	
Ans:	<del></del>	
	Assume beam is simply supported beam and calculate support Reactions.	
	$\sum M_A = 0$ Clockwise moment positive and Anti clockwise moment Negative	
	$-R_B \times 6 + 20 \times 2 + 32 \times 4 = 0$	
	$R_B = 28 \text{ kN}$	
	$R_A + R_B = \text{Total load} = 20 + 32 = 52$	
	$R_A + 28 = 52$	
	$R_A = 24 \text{ kN}$	
	Calculate BM at C and D for simply supported beam	04.7.
	$M_c = 24 \times 2 = 48 \text{ kN.m}$ and moment at D $M_D = 24 \times 4 - 20 \times 2 = 56 \text{ kN.m}$	01 M
	Calculate Fixed End Moments	



	2						
N	$M_A = M_{A1} + M_{A2} = -\frac{W_1 a_1 b_1^2}{I_2^2} - \frac{W_2 a_2 b_2^2}{I_2^2}$						
	Z L						
	$= -\frac{20x2x4^2}{6^2} - \frac{32x4x2^2}{6^2} = 17.78-14.22$	04 3 4					
	o o	01 M					
	$M_A = -32.0 \text{ kN.m}$						
M	$M_{\rm B} = M_{\rm B1} + M_{\rm B2} = -\frac{W_1 a_1^2 b_1}{L^2} - \frac{W_2 a_2^2 b_2}{L^2}$						
$20x2^2x4  32x4^2x2$							
$= -\frac{20x2^2x4}{6^2} - \frac{32x4^2x2}{6^2} = -8.89-28.44$ $M_B = -37.33 \text{ kN.m}$							
							D
	ther						
	20KN 32KN						
	$A = \frac{\sqrt{C}}{2m} + \frac{\sqrt{D}}{2m} + \frac{D}{2m} = \frac{D}{2m}$						
1	$A = \frac{1}{2m} = \frac{1}{$						
	48 KN M 56 KN M						
	3723 KOLM						
	32 kN·m						
	- V-ve/						
	FINAL BMD						
C	Calculate fixed end moments and Draw BMD for a beam as shown in Fig. No. 5.						
	Jse first principle method.						
<b>c</b> )	9.6 k N/m						
	A Jammann B						
	5m						
	1 Assuma has in simply and the same of the						
Ans:	1. Assume beam is simply supported beam and calculate simply supported BM.						
N	$M \max = M_{AB} = \frac{wL^2}{8} = \frac{9.6x5^2}{8} = 30.0kN.m$	01 M					
	2. Calculate Fixed end Moments						
	$M_A + M_B = \frac{-2a}{L}$						
	a = Area of SS BM dia. = area of Parabola = $2/3$ bh						
	a = A1ea of SS BM dia. = a1ea of Parabola = 2/3 bif $a = 2/3 \times 5 \times 30 = 100 \text{ kN.m}$						
	141V 1141R — 2 - 40 (1)						
	$M_A + M_B = \frac{-2x100}{5} = -40$ (I)						

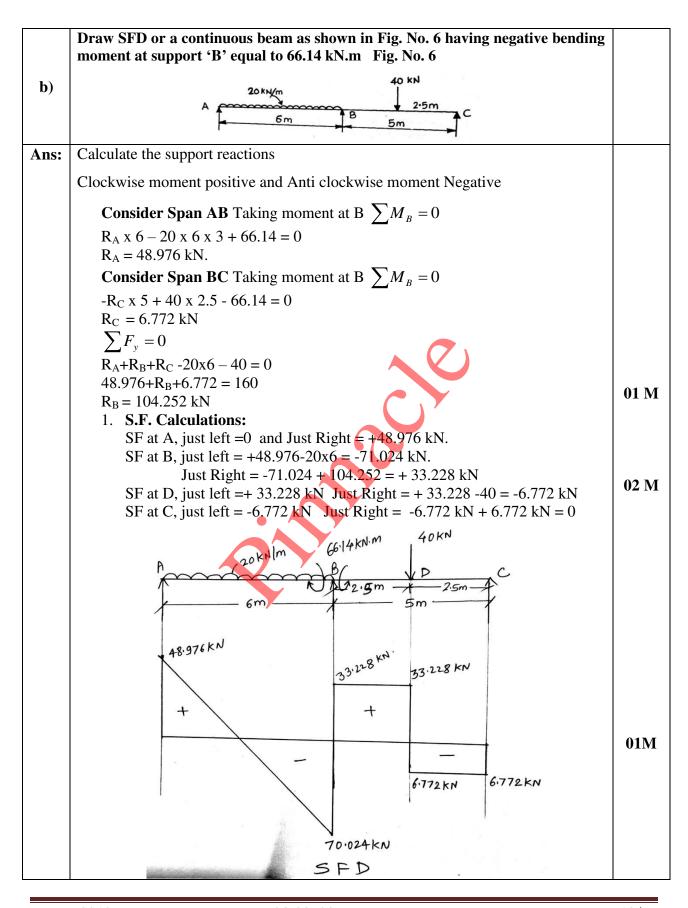


	and $M_A+2$ $M_B=\frac{-6ax}{L^2}$ $x = C.G.$ of SS BM = $5/2 = 2.5m$ $M_A+2$ $M_B=\frac{-6x100x2.5}{5^2}=-60$ (II)  Solving Two Simultaneous Equations I and II $M_A = -20$ kN.m $M_B = -20$ kN.m  OR  Note: Fixed end moments can be calculated by using standard formula as formula is Derived using First Principle, hence if students solve problem using formula appropriate Marks shall be given $M_{AB} = -\frac{wL^2}{12} = -\frac{9.6x5^2}{12} = -20.0kN.m$ $M_{BA} = \frac{wL^2}{12} = +\frac{9.6x5^2}{12} = +20.0kN.m$ 3. Draw Final BM diagram by overlapping simply supported BM and Fixed	01 M 01 M
	end BM.  Sm  Sokn.m  20kn.m  20kn.m  20kn.m  Explain with sketch the effect of fixity on bending moment of a beam.	01 M
d) i) Ans:	If simply supported beam is considered subjected to any pattern of loading, beam bends and slopes will developed at the ends. If however, the ends of beam is firmly built in supports i.e. ends are fixed, slopes at the supports are zero. Fixity at ends induces end moments. Due to fixity, deflection of beam at center of beam is also reduced as compared to simply supported beam.	01 M
	Simply supported beam  W1  W2  MA  Fixed Beam	01 M



(ii)	State two advantages of fixed beam over simply supported beam.					
	1. End slopes of fixed beam are zero					
	2. A fixed beam is more stiff, strong and stable than a simply supported beam.					
	3. For the same span and loading, a fixed beam has lesser values of bending					
Ans:		02 M for				
	moments as compared to a simply supported beam.	any 2				
	4. For the same span and loading, a fixed beam has lesser values of					
	deflections as compared to a simply supported beam.					
Q.4.	Attempt any THREE of the following	12				
a)	State Clapeyron's theorem of three moments for continuous beam with same					
Ans:	and different EI  The claperon's theorm of three moment is applicable to two span continuous beams. It state that "For any two consecutive spans of continuous beam subjected to an external loading and having uniform moment of inertia, the support moments M <sub>A</sub> , M <sub>B</sub> and M <sub>C</sub> at supports A,B and C respectively are given by following equation	01 M				
	given by following equation $M_A + 2M_B(L_1 + L_2) + M_C L_2 = -\left[\frac{6A_1X_1 - L_2}{L_1}\right] - \left[\frac{6A_2X_2}{L_2}\right]$	01 M				
	If the moment of inertia is not constant then claperon's theorem can be stated in the form of following equation.	01 M				
	$M_A \frac{L_1}{I_1} + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2}\right) + M_C \frac{L_2}{I_2} + M_C \frac{L_2}{I_2} = -\left[\frac{6A_1X_1}{L_1I_1} + \frac{6A_2X_2}{L_2I_2}\right]$ Where L <sub>1</sub> and L <sub>2</sub> are length of span AB and BC respectively.  I <sub>1</sub> and I <sub>2</sub> are moment of inertia of span AB and BC respectively.  A <sub>1</sub> and A <sub>2</sub> are area of simply supported BMD of span AB and BC respectively.  X <sub>1</sub> and X <sub>2</sub> are distances of centroid of simply supported BMD from A and C respectively.	01 M				







	Calculate distribution factors for the members OA, OB, OC and OD for the joint 'O' as shown in Fig. No. 7.							
	Joint C	o do silo wii	m 1 ig. 1 (0. 7)	B				
c)				3m (2)				
			A 4m 2D	0 3m C				
	2m 🗊							
Ama	Taint	Mamban	Ct:ffrage Feeter	D Tatal atiffeess	Distribution			
Ans:	Joint	Member	Stiffness Factor	Total stiffness	Distribution Factor			
		OA	$\begin{array}{c c} K_{OA} = \\ 4EI & 4E(2I) \end{array}$	$\sum K_o = 2EI +$	$DF_{OA} = \frac{2EI}{7EI}$			
			$\frac{4EI}{L} = \frac{4E(2I)}{4}$	2EI + 3EI $= 7EI$	DF <sub>OA</sub> =0.286	01 M for		
			= 2 <i>EI</i>	. 7		each D.F.		
		OB	$K_{OB} = \frac{3EI}{L}$		$DF_{OB} = \frac{2EI}{7EI}$	<b>D.F.</b>		
	О		$=\frac{3E(2I)}{3}=2EI$	70'	$DF_{OB} = 0.286$			
		OC	$K_{OC} = \frac{3EI}{L}$		3 <i>EI</i>			
			_		$DF_{OC} = \frac{3EI}{7EI}$			
			$=\frac{3E(3I)}{3}=3EI$		DF <sub>OC</sub> =0.428			
		OD	K <sub>OD</sub> = 0		DF <sub>OD</sub> =0			
	Calculate support moments and Draw BMD of a beam as shown in Fig. No. 8. Use moment distribution Method.							
		-	20 KN	30 I	KN			
<b>d</b> )			A 3m	mm B	3			
				7. 7				
Ans:	Fig. No. 8							
	1. Calculate simply supported BM for span AB $m_{AB} = \frac{wL^2}{8} = \frac{20x3^2}{8} = 22.5kN.m$							
		$m_{AB} - \frac{1}{8}$	- = - 22.3kIV	v .111				
			Fixed end Moment for	or span AB				
	$M_{AB} =$	$-\frac{wL^2}{12} = -$	$\frac{20x3^2}{12} = -15kN.m$					



	$M_{BA} = \frac{wL^2}{12} = +\frac{20x3^2}{12}$	! - = +	-15 <i>kN.m</i>		01 M
	$M_{BC} = -30 \text{ x } 1 = -30 \text{ k}$ Distribution factor		$DF_{BA} = 1.0, DF_{BC} = 0$	as it is overhang  Joint	
	AB	BA	BC CB	Member	
		1.0	0	Distribution factor	<b>7</b>
	-15 +	15	-30 0	Fixed end moments	Table 02 M
		<b>⊦</b> 15		Balancing at B	
	+7.5			Carryover to A	
	-7.5 +	30	-30 0	Final Moments	
	7.5 KN m		22.5 KN·m 30 KN·r Ve	m — m	01 M
(i)	Draw one Sketch of the Deficient frame	ie to	Howing.		
Ans:					01M
(ii) Ans:	Redundant frame				
AHS.		<u> </u>			01 M



(iii)	Symmetrical portal frame	
Ans:	w/unit length W	
	Rigid joint L <sub>2</sub> Rigid joint L <sub>2</sub> Rigid joint L <sub>3</sub> (i) Symmetrical portal frame fixed at the base (ii) Symmetrical portal frame simply supported (hinged) at the base	Any 01 one mark
(iv)	Unsymmetrical portal frame	
Ans:	(i) Unsymmetrical portal frame hinged at the base  (ii) Unsymmetrical portal frame one end fixed, other hinged  Note- Other than these above sketches if any relevant sketch is drawn, the marks are given accordingly.	Any 01 one mark
Q.5.	Attempt any TWO of the following	12 M
Ans:	Calculate Maximum Deflection of Simply Supported Beam as Shown In Fig no-9.  take E=200gpa I=2x 10 Use Macaulay's Method.	
Alls:	E=200 GPA =200x10 <sup>3</sup> =N/mm <sup>2</sup> E=200x10 <sup>3</sup> =2x10 <sup>8</sup> KN/m <sup>2</sup> I=2x10 <sup>8</sup> = mm <sup>4</sup> I=2x10 <sup>-4</sup> m <sup>4</sup> 1)Find support Reaction	01 M



RA = RB = W1/2 = 20X3/2 = 30KN	
2)Find slope &deflection	
EI $d^2y/dx^2 = M$ -Differential equation	
Taking moment at section X-X, and at distance x from A	
$EI d^2 y / dx^2 = 30x - 20x^2 / 2$	
•	
$EI d^2y/dx^2 = 30x \left  10x^2 \right $	
Integrating w. r to x	
EI dy /dx = $30 \times \frac{2}{2} + C1 \left  -10 \times \frac{3}{3} \right $	
EI dy/dx= $15x^2$ +C1 $\left  -3.33x^3 \right $ slope equation	01 M
stope equation	
Again integrating w.r to x	
EIy= $15x^3/3+C1x+C2-3.33x^4/4$	
Ely= $5x^3$ +C1x + C2 -0.832 x <sup>4</sup> Deflection equation	
Ely=5x +C1x + C2 -0.832 x Deflection equation	01 M
To find C2	
Boundary condition	
x=0 Y=0 put in <b>Deflection Equations</b> .	
E1(0) = 5(0) + c1(0) + c2 - 0.83(0)4	
C2=0	
To find C1	
Boundary condition	
At x=3 y=0 put in deflection equation	
0=05(3)3+c1x3+0-0.832*3	
3C1=67.608	
C1= -22.53	01 M
Put this value in Deflection equation	
$EIy = 5x^{3} - 22.53 \times -0.832x^{4}$	
To find Maximum Deflection	
Put $x=L/2 = 3/2 = 1.5 \text{ m}$	
$EIY = 5(1.5)^3 - 22.53 * 1.5 - 0.832(1.5)^4$	
EIY= -21.132	
	01 M
$E=200 \text{ GPA} = 200 \times 10^3 = \text{N/mm}^2$	
$E = 200 \times 10^3 = 2 \times 10^8 \text{ KN/m}^2 \text{ (note:- W is in KN/m and L is in m.)}$	
$E = 200 \times 10 = 2 \times 10 \text{ KN/m} \text{ (note:- W 1S 1n KN/m and L 1S 1n m.)}$	



	8 4	
	$I=2x10^8 = mm^4$	
	$I=2x10^{-4}m^4$	
	Y= -21.132/EI -4 8	
	$= 21.132/(200x10^{-4} * 200x10^{8})$	
	Y max = 0.0005288	01 M
<b>L</b> )	Y max=0.528 mm ( - ve indicate downward deflection)  Calculate Maximum Slope & Maximum Deflection Of A Cantilever Beam As	
<b>b</b> )	Shown In Fig	
	A Jammmmm B	
	2m	
Ans:	Given :-	
	$E=100  ext{ GPA}=100  ext{X} 10^3  ext{ N/mm}^2$	
	Width =100 mm ,depth=200mm	
	$I=bd^3/12 =100*(200)^3/12 \neq 66.66\times10^6$	
	Maximum deflection =Deflection due to UDL+ deflection due to point load	
	YB=yB1+yB2	
	$Yb1=-WL^4 / 8EI = (-2X(2000)^4) / (8X100 * 10^3 * 66.66X10^6)$	
	=-0.600 mm	1M
	$Yb2 = -WL^{4}/3EI = (-5000X(2000)^{3})/(3*100*66.66*10^{6}*10^{3})$	
	= -2.01 mm	1M
	YB = YB1+YB2 = -(0.6+2.01) = -2.6  mm	1 M
	maximum slope = slope due to UDL + slope due to point load	
	$\theta = \theta + \theta 2$	
	$\theta 1 = W L^{3} / 6 EI = (2*2000^{3} / 6*100*10^{3} * 66.66 X 10^{6})$	
	=0.0004 Radian	1M
	$\theta 2 = W L^2 / 2 EI = (5000*2000^2 / 2*100 X 10^3 * 66.66*10^6)$	
	$62 = \text{W L } / 2 \text{ E1} = (3000^{\circ} 2000 / 2^{\circ} 100 \text{ A} 10^{\circ} \circ 60.00^{\circ} 10^{\circ})$ =0.0015 Radian	1M
	$\theta = 0.0004 + 0.0015 = 0.0019$ Radian	
	deflection Maximum =2.6mm ( -ve indicates the downward deflection )	
	Maximum slope =0.0019 Radian	1 M
<u> </u>		# 17#

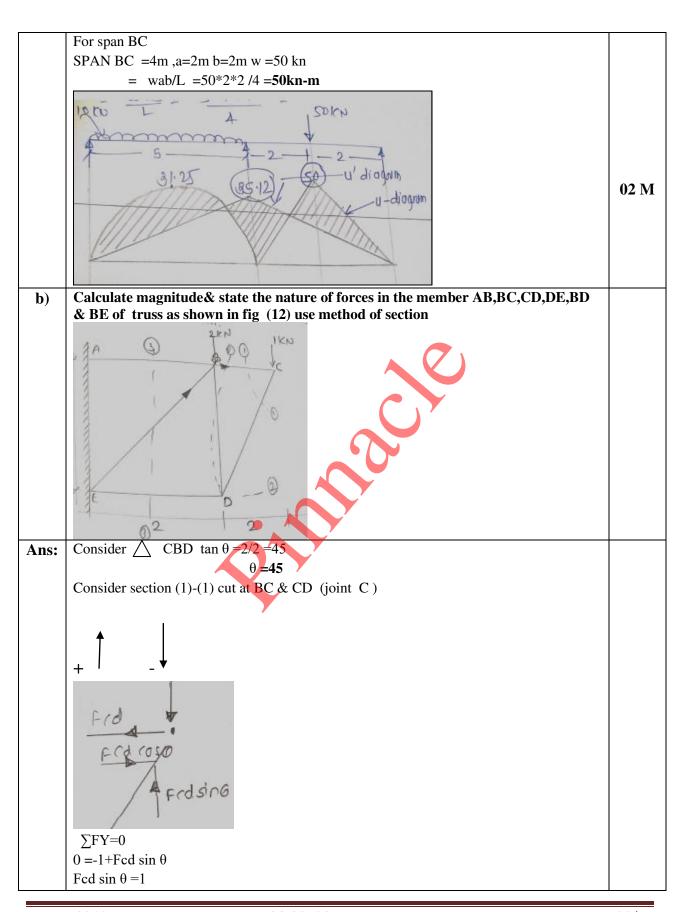


5)	Calculate Support Moments For A Doom As Chaum In Fig No A0	
c)	Calculate Support Moments For A Beam As Shown In Fig No-08.  Use Three Moment Theorem.	
	Ose Three Moment Theorem.	
	(20K10/m   30KN	)
	10 1 00 x B	
	To A ferrammy Ac	
	1 3 m 1 1 m k	
	1-10-14	13
	1 1 1	
Ans:	TO find support moments and reactions	
	B.M at mid span AB = $WL^2 / 8 = 20(3)^2 / 8$	
	= 22.5  KN.M	
	Consider the cantilever action point BC	
	MB = -30X1 = -30KNm	
	Since the end A is fixed assume as imaginary span A-AO at left of A	01 M
	For span AO - A	
	6  ao* o  / L0 = 0	
	Span AO A B	
	A1 = Area Of A Diagram = $(2/3) * 3 * 22.5 = 45$	
	X1 = centroidal distance of a diagram = $3/2$ = $1.5$ m	
	A1 X1 = 45*1.5 = <b>67.5</b>	01 M
	Applying clapeymn's theorem of three moment for span A Ao & AB we get	0435
	Mo L0+2MA (Lo+L1) +MBL1 =- $[6a0X0/Lo + 6a1x1/L1]$	01 M
	0+2MA (0+3) + (-30) (3) = [0+6X67.5/3]	
	6 MA -90 =-135	01 M
	6 MA = -135+90=-45	UI WI
	MA=-7.5 KN-m	
	Consider Span ABC	
	A (20KN/m 30	
	1 Zot NIR)	
	<b>4</b>	
	7.5 7 9K'Y	
	130	
	Take moment @ a	
	O=20*3*1.5+30+30*4 -RB*3	01 1/4
	RB*3 = 240 $RB=80KN$ .	01 M
	$\sum$ fy = 0	
	0 = RA + RB - 20X3 - 30	
	0=RA+80-60-30	01 M
	RA = 10KN	V 111
		<u>I</u>



2.6.	Attempt Any Two of the fo	ollowing		12 M
a)	calculate support moment	for a spam as shown in	fig no.11 Use moment	
	distribution method			
	10 KN	Im .		
	(	SOKN		
	Ammin	2m C		
	(2I)	4 (I)		
	5 m	400		
ns:	Solution :- Assume span AB &	BC as a fixed beam and find	fixed end moment	
	$M AB = -WL^2/12 = -10(5)^2/$			
	_			
	$M BA = WL^2/12 = 10(5)^2/12$			
	$M BC = -Wab^2/L^2 = 50(2) (2)$	$\frac{2}{4^2} = -25$ KN-m		01 N
	$M CB = + Wab^2/L^2 = 5*2*2^2$	_		01 N
			•	
	To find the Stiffness factor a $K BA = 3EI/L AB = 3E(2I)/5$ :			
	K BC = 3EI/LBC = 3EI/4 = 0			
	$\Sigma$ K=1.2EI+0.75EI= <b>1.95</b> EI	.73 EI		
	Distribution Factor			
	DFBA=KBA/ $\Sigma$ K =1.2EI/1.95	EI=0.62		01 N
	DFBC = KBC/ $\Sigma$ K =0.75EI/1.9			
	Point	A B	С	
	Member	AB	BC CB	
	1120111002	BA		
	Distribution factor	0.62	0.38	
	Fixed end moment	-20.83	-25 25	
		20.83		
	Release support A& C	<del>-                                    </del>	-25	
	and then carry over from	+20.83		
	A to B from C to B			0.0.3
			-12.5	02 N
		10.415		
	Initial moment	0	-37.5	
		31.245		
	Ist distribution C balance	+3.87	+2.37	
	В			
	Final moment	+35.12	-35.12	
	Assume span AB and BC to b	e simply supported beam and fi	nd free BM.	
	-	0KN/m		
	M max =w1 $^2$ /8 =10*(5) $^2$ /8 =			
	1VI III A - WI / O - IU (3) / O -	-01,20 IXIVIII		





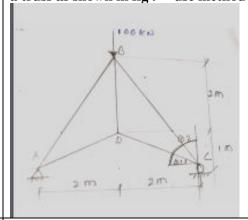


End-1 /1 IZM (C)			
Fcd=1.41 KN (C)			
$\Sigma$ FX=0			
$0 = \text{Fcd } \cos \theta - \text{Fcb}$			
Fcb = Fcd $\cos \theta$			
$= 1.41 \cos 45$			
=0.997			02 M
= 1KN(T)			02 111
= IKN(I)			
Consider section (2)-(2) of	cut at CD,BC,ED		
Consider right hand side	, ,		
Fedsir	ra		
FBd &	1 =		
fed o	7010		
C 1	4	<b>7</b>	
Fed D			
	<b>F</b> 6 0		
	$\sum_{n=1}^{\infty} fy = 0$		02 M
	$0=-1$ -Fcd sin $\theta$ +		02 111
	Fbd=1+Fcd sin	15	
	Fbd=2(T)		
$\sum fx=0$	Y		
$0 = - \text{Fcd } \cos \theta + \text{Fed}$			
$1.41 \cos 45 = \text{Fed}$			
Fed =1.41 cos 45			
Fed=1 kN(c)			
Consider section	(2) (2) take manual at (2)	Δ	
	1 (3)-(3), take moment at @	A	
0=Fbe cos 45 +F		A	
0=Fbe cos 45 +F 10 = 1.41 Fbe	ed *2 +2*2+1*4	A	0135
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v		A	01 M
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum fx=o$	ed *2 +2*2+1*4 /e indicate compressive)	A	01 M
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum$ fx=o 0= -fab+feb cos4	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed	A	01 M
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum$ fx=o 0= -fab+feb cos4 Fab =7.092X CO	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed	A	
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum$ fx=0 0=-fab+feb cos4	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed	A	01 M 01 M
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum$ fx=o 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T)	ed *2 +2*2+1*4  /e indicate compressive)  5 +fed  9S 45 +1		
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v $\sum$ fx=0 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T)	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed 0S 45 +1  FORCE (KN)	NATURE	
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v ∑fx=o 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T) MEMBER AB	ed *2 +2*2+1*4  /e indicate compressive)  5 +fed  9S 45 +1	NATURE TENSION	
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v ∑fx=o 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T) MEMBER AB BC	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed 0S 45 +1  FORCE (KN) 6.014 1	NATURE TENSION TENSION	
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v  ∑fx=o 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T)  MEMBER  AB BC CD	ed *2 +2*2+1*4  ve indicate compressive)  5 +fed 0S 45 +1  FORCE (KN)	NATURE TENSION TENSION COMPRESSION	
0=Fbe cos 45 +F 10 = 1.41 Fbe F be =7.092 (-v ∑fx=o 0= -fab+feb cos4 Fab =7.092X CO Fab =6.014 (T) MEMBER AB BC	ed *2 +2*2+1*4  /e indicate compressive)  .5 +fed  .S 45 +1  FORCE (KN)  6.014  1 1.41	NATURE TENSION TENSION	



c) calculate magnitude &state the nature of forces in member AB,BC,CD,AD&BD Of a truss as shown in fig. use method of joints.

 $\theta 2 = \tan \theta \ 2 = 3/2$ 



Ans:  $\sum fy = 0$ 

RA+RC =100, due to symmitricity

RA=RC=W/2=100/2=50KN

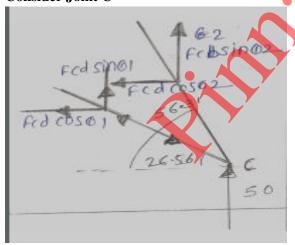
Consider joint C

 $\theta$  1 = tan  $\theta$  1 = 1/2

 $\theta$  1 = 26. 56  $\theta$ 2 = 56.31

01 M

Consider Joint C



 $\sum$ Fx= 0 fcd cos  $\theta$  1 +fcb cos  $\theta$  2= 0 0.8944 fcd +0.55fcb= 0

 $\sum$  fy =0

 $0=50+\text{fcd sin }\theta 1+\text{fcb sin }\theta 2$ 

 $-50 = \text{fcd sin } \theta 1 + \text{fcb sin } \theta 2$ 

-50=0.4471fcd+0.832 fcb



